

## Tutorial on Mathematical Modeling

A-2-4. Figure 2-37(a) shows a schematic diagram of an automobile suspension system. As the car moves along the road, the vertical displacements at the tires act as the motion excitation to the automobile suspension system. The motion of this system consists of a translational motion of the center of mass and a rotational motion about the center of mass. Mathematical modeling of the complete system is quite complicated.

A very simplified version of the suspension system is shown in Fig. 2-27(b). Assuming that the motion  $x_i$  at point  $P$  is the input to the system and the vertical motion  $x_o$  of the body is the output,

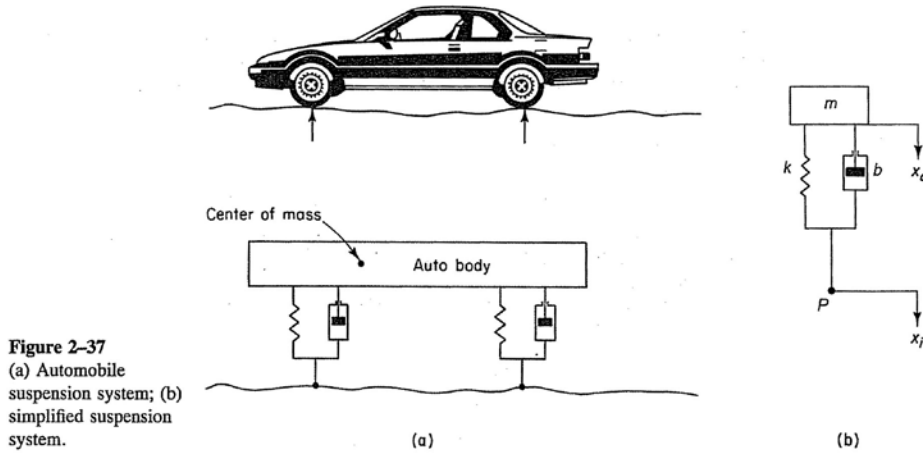
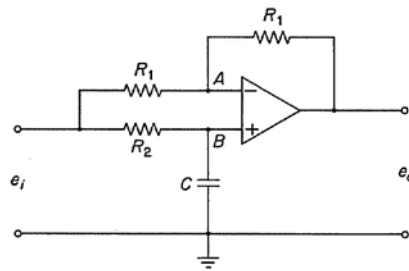


Figure 2-37  
(a) Automobile suspension system; (b) simplified suspension system.

obtain the transfer function  $X_o(s)/X_i(s)$ . (Consider the motion of the body only in the vertical direction.) Displacement  $x_o$  is measured from the equilibrium position in the absence of input  $x_i$ .

Q2 Obtain the transfer function of the circuit



Q3

A-2-11. Obtain the closed-loop transfer function for the positional servo system shown in Figure 2-45. Assume that the input and output of the system are the input shaft position and the output shaft position, respectively. Assume the following numerical values for system constants:

- $r$  = angular displacement of the reference input shaft, radians
- $c$  = angular displacement of the output shaft, radians
- $\theta$  = angular displacement of the motor shaft, radians
- $K_1$  = gain of the potentiometric error detector =  $24/\pi$  volts/rad
- $K_p$  = amplifier gain = 10 volts/volt
- $e_a$  = applied armature voltage, volts
- $e_b$  = back emf, volts
- $R_a$  = armature-winding resistance = 0.2 ohms
- $L_a$  = armature-winding inductance = negligible
- $i_a$  = armature-winding current, amperes
- $K_b$  = back emf constant =  $5.5 \times 10^{-2}$  volts-sec/rad
- $K$  = motor torque constant =  $6 \times 10^{-5}$  N-m/ampere
- $J_m$  = moment of inertia of the motor =  $1 \times 10^{-5}$  kg-m<sup>2</sup>
- $b_m$  = viscous-friction coefficient of the motor = negligible
- $J_L$  = moment of inertia of the load =  $4.4 \times 10^{-3}$  kg-m<sup>2</sup>
- $b_L$  = viscous-friction coefficient of the load =  $4 \times 10^{-2}$  N-m/rad/sec
- $n$  = gear ratio  $N_1/N_2 = 1/10$

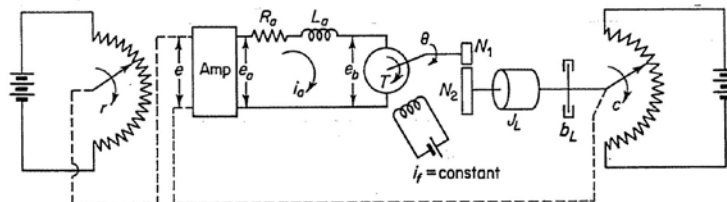


Figure 2-45  
Positional servo system.